

Figure 1: Schematics of the 5 cm WSB used in the LBNL water heater simulator ( $A_c / A_b = 15$ ).

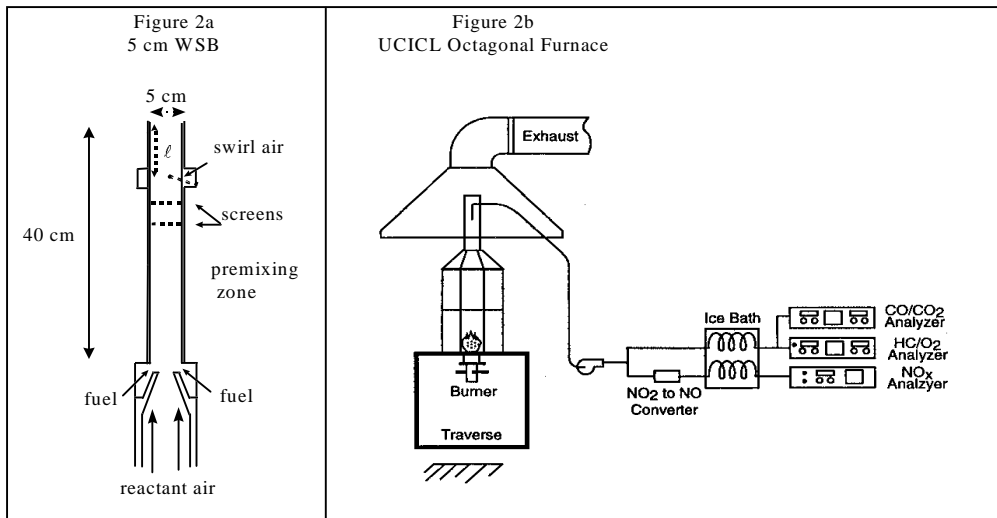


Figure 2: Schematics of the 5 cm WSB used in the UCICL octagonal enclosure ( $A_c / A_b = 142$ ).

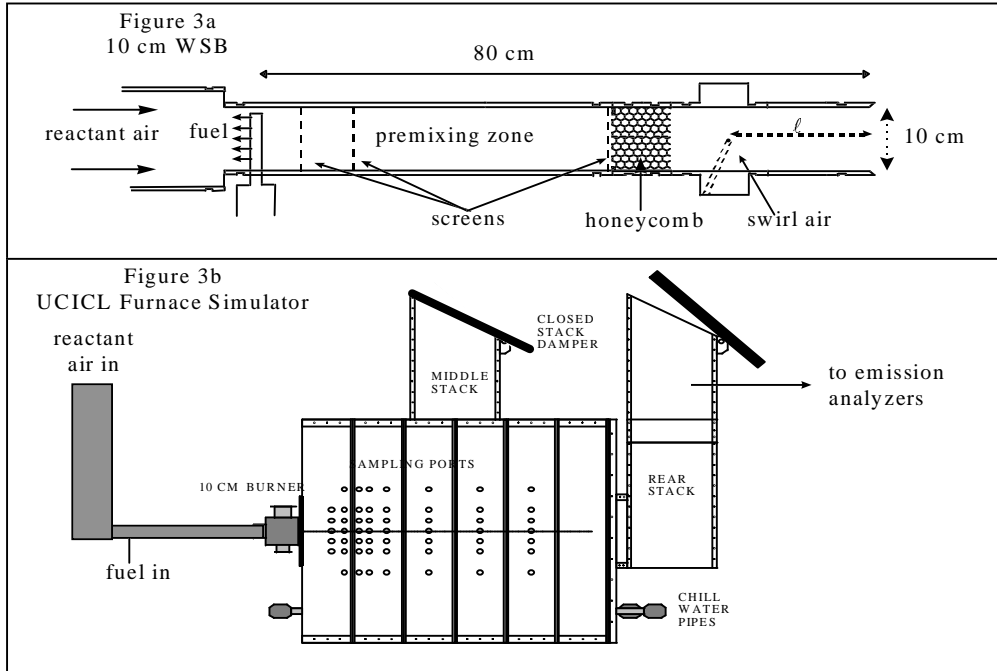


Figure 3: Schematics of the 10 cm WSB in the UCICL furnace simulator ( $A_c / A_b = 733$ ).

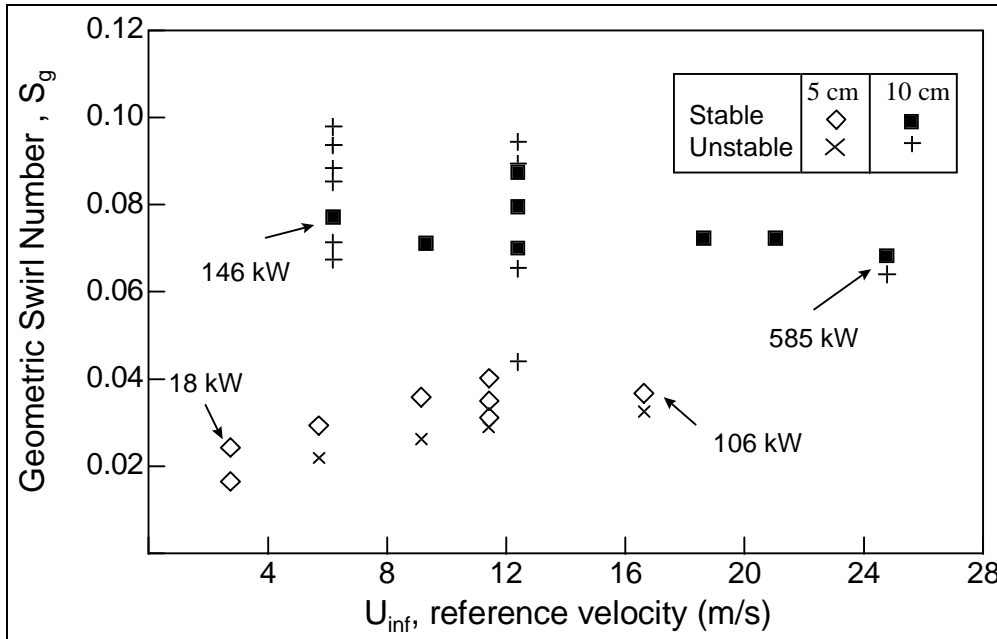


Figure 4: The operating regime of the 5 and 10 cm in the UCICL combustion chambers over similar velocity regimes shows a significant increase in the non-dimensional swirl number  $S_g$  (and thus swirl air) needed for stability.

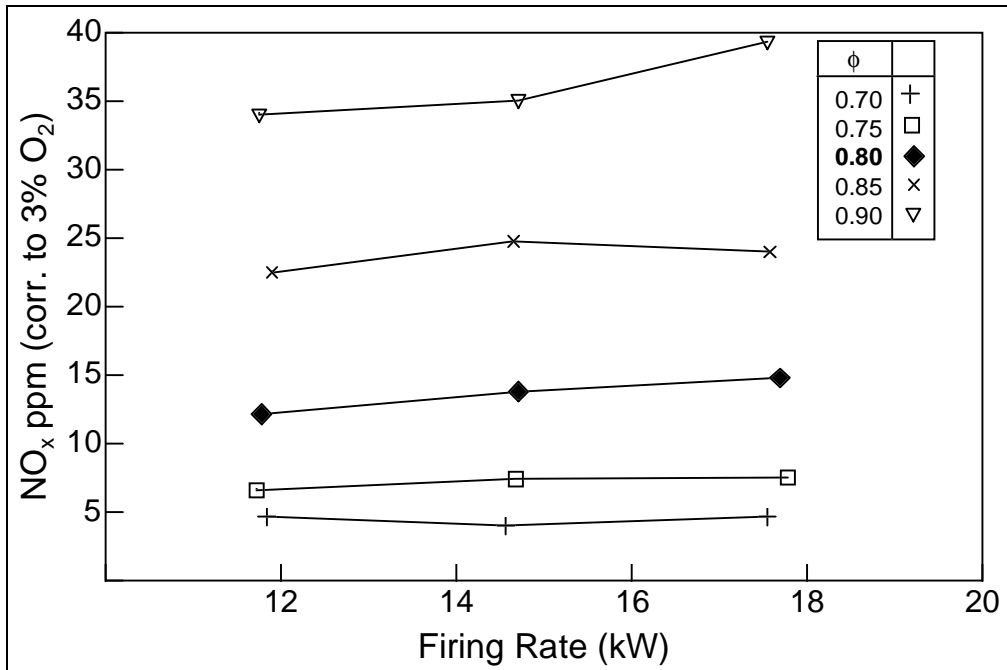


Figure 5:  $\text{NO}_x$  emissions from the LBNL water heater simulator with the 5 cm WSB ( $A_c / A_b = 15$ ) displays primary dependence on the equivalence ratio and little dependence on firing rate. For  $\phi < 0.90$ , the WSB exceeds the strictest regulations of SCAQMD 1146.2

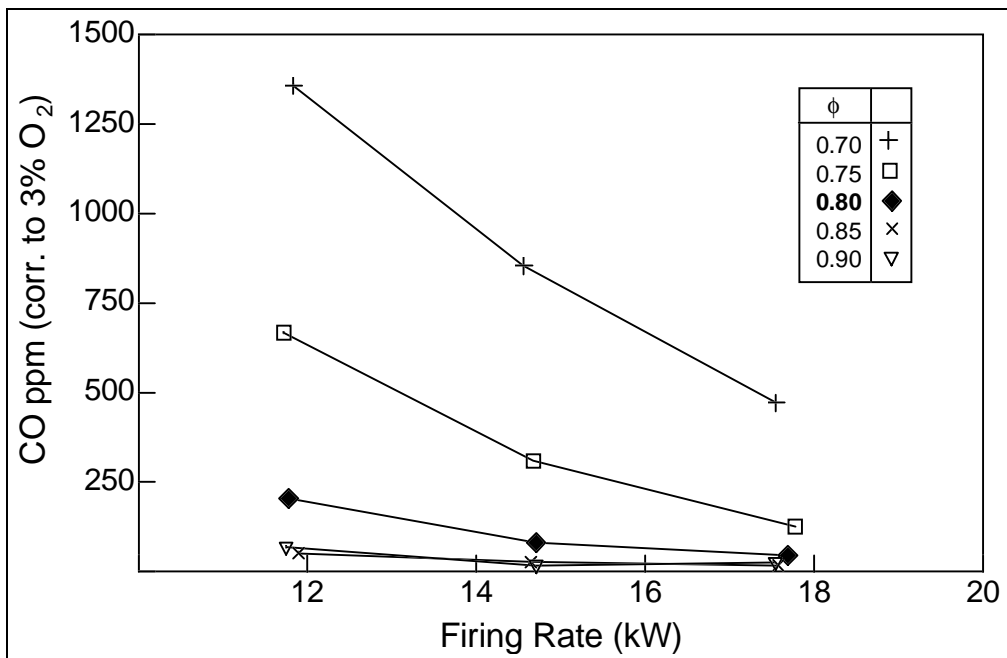


Figure 6: CO emissions from the LBNL water heater simulator with the 5 cm WSB ( $A_c / A_b = 15$ ) demonstrate dependence on both firing rate and equivalence ratio.

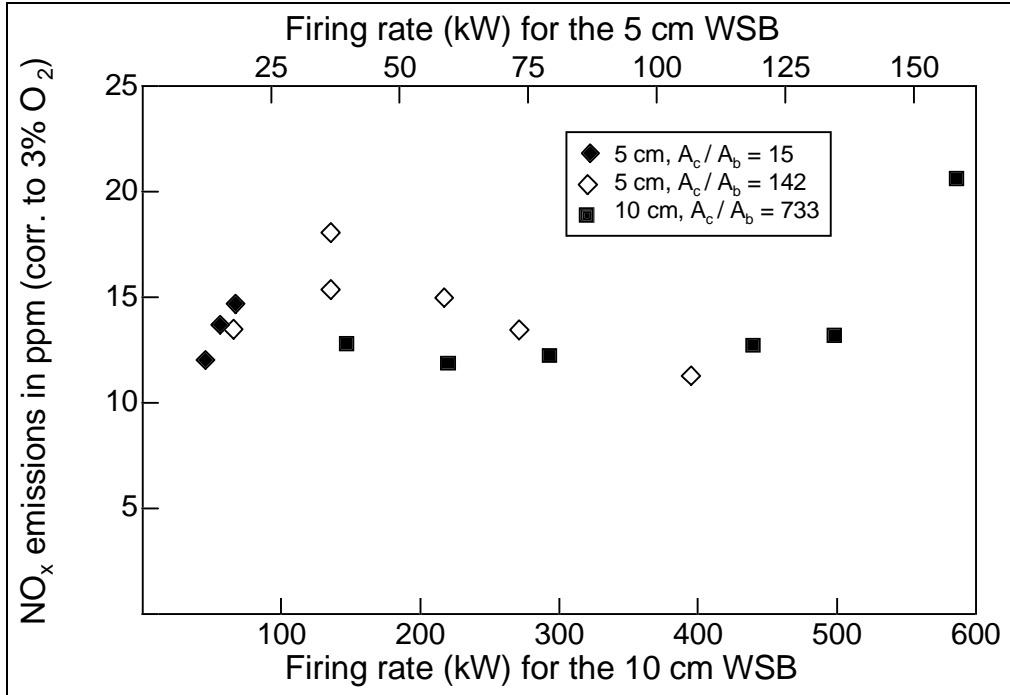


Figure 7: NO<sub>x</sub> levels are constant at  $\approx 15$  ppm for the three chamber/burner ratios and for a firing rates from 12 to 585 kW.

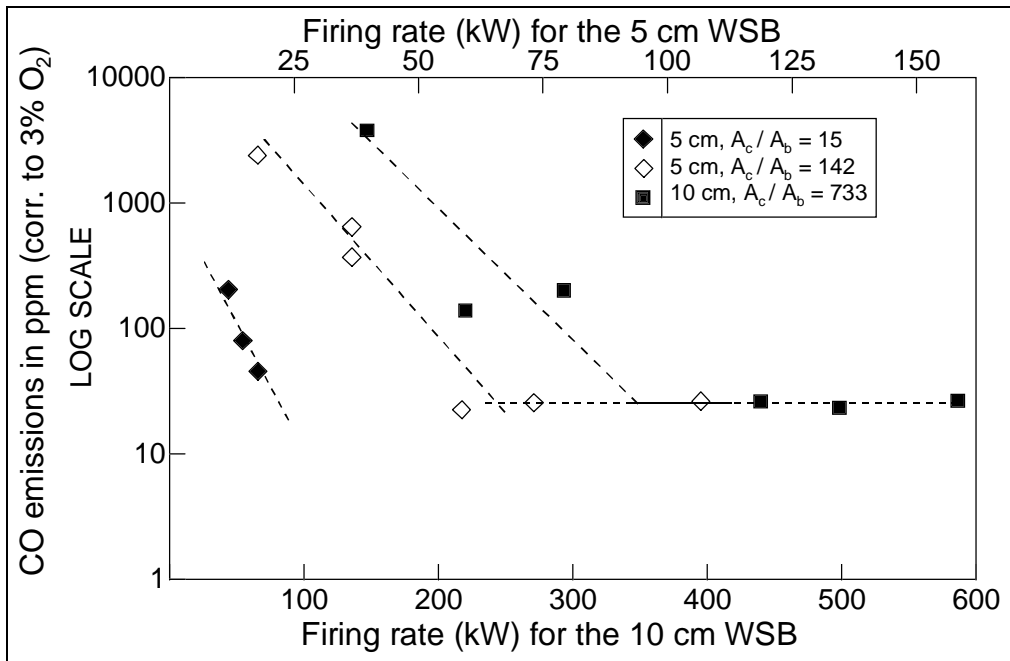


Figure 8: As plotted on a logarithmic scale, CO emissions display a strong dependence on both firing rate and  $A_c / A_b$ . Minimum firing rates of  $\approx 25$ , 65, and 400 kW are necessary to achieve CO levels of 25 ppm for  $A_c / A_b = 15$ , 142, and 733 respectively. Dashed lines are not fit to the data.

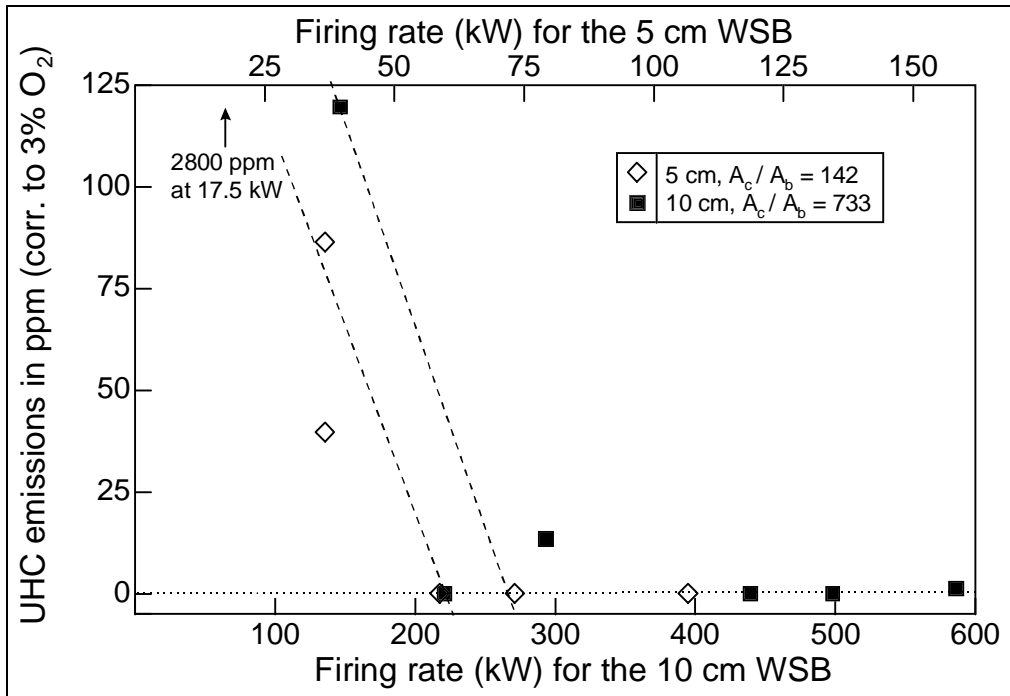


Figure 9: UHC emissions decrease dramatically with increasing firing rate however no conclusion can be drawn on the effect of  $A_c / A_b$  on emission levels. Above  $\approx 75$  and  $300$  kW (similar  $U_{ref}$  of  $12$  m/s), emissions are essentially  $0$  ppm.